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Control design for partial stabilization of nonlinear mechanical systems with random disturbances^{*}

Zuyev A.L.^{1,2}, Vasylieva I.G.¹

 ¹ Institute of Applied Mathematics and Mechanics, National Academy of Sciences of Ukraine, Slovyansk; irisna.shurko@gmail.com,
 ² Max Planck Institute for Dynamics of Complex Technical Systems, Magdeburg, Germany; zuyev@mpi-magdeburg.mpg.de.

The problem of the partial stabilization of nonlinear control systems described by the Ito stochastic differential equations is considered. For these systems, we propose a constructive control design method, which provides the partial asymptotic stability in probability of the trivial solution of the closed-loop system with respect to a part of state variables. Mechanical examples are presented to illustrate the efficiency of the proposed controllers.

У роботі розглянуто задачу стабілізації щодо частини змінних для нелінійних керованих систем, які описуються стохастичними диференціальними рівняннями Іто. Запропоновано конструктивний метод побудови функцій зворотного зв'язку, що забезпечують часткову асимптотичну стійкість за ймовірністю тривіального розв'язку відповідної замкнутої системи. Ефективність одержаних законів керування проілюстровано на прикладах механічних систем.

В работе рассматривается задача стабилизации относительно части переменных для нелинейных управляемых систем, которые описываются стохастическими дифференциальными уравнениями Ито. Предложен конструктивный метод построения функций обратной связи, обеспечивающих частичную ассимптотичесую устойчивость по вероятности тривиального решения соответствующей замкнутой системы. Эффективность полученных управлений проиллюстрирована на механических примерах.

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A Introduction

To construct adequate mathematical models that describe the behavior of real dynamic processes and analyze their stability properties, it is necessary to take into account the effects of uncertainties and random disturbances. The latter leads to the need to study systems of differential equations with random perturbations. Here, qualitative methods for investigating the asymptotic behavior of solutions of systems of differential equations with random disturbances are useful. Lyapunov methods for analyzing the stability of stochastic systems have been developed by many authors (see, e.g., 12 and references therein). In particular, the concept of control Lyapunov functions and Artstein's theorem 3 have been extended to stochastic differential equations in 4. In 5, a criterion for stochastic finite-time stability via multiple Lyapunov functions has been obtained.

Partial stabilization problem arises in tasks when only the stability with respect to some variables is needed for a desired performance of the system. This task is also crucial when the system is not stable in the sense of Lyapunov, but asymptotically stable with respect to a part of variables 69. Therefore, the problems of partial stability and stabilization of motion are highly important in engineering applications, cf. 1011. In the paper 9, conditions of partial stability in probability for the Ito stochastic differential equations have been obtained by Lyapunov's direct method. In 12, sufficient conditions for partial stability of stochastic reaction-diffusion systems with Markovian switching have been derived.

In this paper, we consider the problem of stabilization of the Itotype stochastic differential equations with respect to a part of variables. Our goal is to propose an efficient control design scheme for the above problem. To achieve this goal, we present an extension of the universal stabilizing controllers from 13 to the problem of partial stabilization of stochastic systems in Section 3. Our main theoretical contribution will be applied to mechanical examples in Sections 4 and 5.

B Notations and Definitions

Throughout this paper, let $w(t) \in \mathbb{R}^k$ $(t \ge 0)$ be a standard k-dimensional Wiener process defined on a complete probability space (Ω, \mathcal{F}, P) , and let $\{\mathcal{F}_t\}_{t\ge 0}$ be the complete right-continuous filtration generated by w. Consider a control system described by the Ito stochastic differential equations:

$$dx(t) = (f(x) + g(x)u)dt + \sum_{i=1}^{k} \sigma_i(x)dw_i(t),$$
(1)

where $x = (x_1, ..., x_n)^T \in D \subseteq \mathbb{R}^n$ is the state and $u = (u_1, ..., u_k)^T \in U = \mathbb{R}^k$ is the control. We assume that $0 \in D$, $\sigma_i(0) = 0$ for i = 1, ..., k, and the maps $f : D \to \mathbb{R}^n$, $g : D \to \mathbb{R}^{n \times k}$, $\sigma_i : D \to \mathbb{R}^n$ satisfy the Lipschitz condition on every bounded domain $X \subset D$.

For a map $h: D \to U$, h(0) = 0, we introduce the closed-loop system for (1) with the feedback law u = h(x):

$$dx(t) = (f(x) + g(x)h(x))dt + \sum_{i=1}^{k} \sigma_i(x)dw_i(t).$$
 (2)

If h is Lipschitz continuous on every bounded $X \subset D$, then there exists a unique strictly Markov process $x^{\xi,s}(t)$ which is a solution of (2) under the initial condition $x^{\xi,s}(s) = \xi$ (see, e.g., 15). We relate with the control system (1) the operator

$$\mathcal{L}_u = \sum_{i=1}^n (f(x) + g(x)u)_i \frac{\partial}{\partial x_i} + \frac{1}{2} \sum_{i,j=1}^n c_{ij}(x) \frac{\partial^2}{\partial x_i \partial x_j}, \quad [c_{ij}(x)] = \sigma(x)\sigma^T(x).$$

In the sequel, we will study stability of the trivial solution of (2) with respect to the variables $x_1, x_2, ..., x_m$. Denote these variables as $y = (y_1, ..., y_m)^T \in \mathbb{R}^m$ and the rest as $z = (z_1, ..., z_p)^T \in \mathbb{R}^p$, m + p = n, then $x = (y^T, z^T)^T$, $x_0 = (y^T_0, z^T_0)^T$, and $||x|| = (x_1^2 + ... + x_n^2)^{1/2} = (||y||^2 + ||z||^2)^{1/2}$.

We assume also that the solutions of (1) are z-extendable in a closed domain $D = \mathcal{D}_H$, where

$$\mathcal{D}_H = \{ x \in \mathbb{R}^n : ||y(t)|| \le H, \ z \in \mathbb{R}^p \}, \ H = const > 0.$$

It means that if $x(t) \in \mathcal{D}_H$ is a maximal solution of system (1) on $t \in (\tau_1, \tau_2)$ with some admissible control $u \in L^{\infty}(\tau_1, \tau_2)$, then either $||y(t)|| \to H$ as $t \to \tau_2$ almost surely or $\tau_2 = \infty$. This kind of z-extendability assumption is natural in the problems of partial stability [6]; it is usually satisfied for well-posed mathematical models in physics whose trajectories do not blow up in finite time with bounded control.

Let us introduce the standard class of comparison functions \mathcal{K} , whose elements are continuous strictly increasing functions $\alpha : \mathbb{R}^+ \to \mathbb{R}^+$ such that $\alpha(0) = 0$. We will extend the concept of a control Lyapunov function 3.4 [13] 14 to the problem of partial stabilization of stochastic systems as follows.

Definition B.1. A function $V \in C^2(\mathcal{D}_H; \mathbb{R})$ is called a *y*-stochastic control Lyapunov function (*y*-SCLF) for system (1), if there exist $\alpha, \beta_1, \beta_2 \in \mathcal{K}$ such that

$$\beta_1(||y||) \le V(x) \le \beta_2(||y||), \quad \inf_{u \in U} \mathcal{L}_u V(x) \le -\alpha(||y||),$$

for all $x \in \mathcal{D}_H$.

Throughout the text, $B(x; \delta)$ denotes the δ -neighborhood of a point $x \in \mathbb{R}^n$.

Definition B.2. A function $V \in C^2(\mathcal{D}_H; \mathbb{R})$ satisfies the small control property with respect to y if, for any $\epsilon > 0$ and any $x_0 \in M = \{x | y = 0\}$, there exists a $\delta > 0$ such that

$$x \in B(x_0; \delta) \Rightarrow \inf_{\substack{||u|| < \epsilon}} \mathcal{L}_u V(x) \le -\alpha(||y||)$$

Definition B.3. 9 16 18 The solution x = 0 of system (2) is called *y*-stable in probability if, for all $s \ge 0, \varepsilon > 0, \gamma > 0$, there exists a $\delta > 0$ such that $\xi \in B(0; \delta)$ implies

$$P\{\sup_{t\geq s}||y^{\xi,s}(t)||>\varepsilon\}<\gamma.$$

Definition B.4. [17] [18] The solution x = 0 of system (2) is called asymptotically y-stable in probability if it is y-stable in probability and

$$P\{\lim_{t \to \infty} ||y^{\xi,s}(t)|| = 0\} = 1$$

for all $\xi \in B(0; \Delta)$ with some constant $\Delta > 0$.

C Main result

The following result generalizes the constructive proof of Artstein's theorem 13 for the problem of partial stabilization of stochastic systems. **Theorem C.1.** Let $V \in C^2(\mathcal{D}_H; \mathbb{R})$ be a y-SCLF satisfying the small control property. Then there exists a continuous feedback law $h : \mathcal{D}_H \to$ \mathbb{R}^k , h(0,z) = 0, such that the trivial solution of the corresponding closedloop system (2) with u = h(x) is y-asymptotically stable in probability. The feedback law h(x) is given as follows:

$$h_{i}(x) = \begin{cases} 0, & b = 0, \\ -\frac{b_{i}}{\|b\|^{2}} (a + (a^{2} + \|b\|^{4})^{\frac{1}{2}}), & b \neq 0, 2(a^{2} + \|b\|^{4})^{\frac{1}{2}} \ge \alpha(\|y\|), \\ -\frac{b_{i}}{2\|b\|^{2}} (2a + \alpha(\|y\|)), & otherwise, \end{cases}$$
(3)

where

$$a(x) = \sum_{i=1}^{n} f_i(x) \frac{\partial V(x)}{\partial x_i} + \frac{1}{2} \sum_{i,j=1}^{n} c_{ij}(x) \frac{\partial^2 V(x)}{\partial x_i \partial x_j},$$

$$b_i(x) = \sum_{j=1}^{n} g_{ij}(x) \frac{\partial V(x)}{\partial x_j}, \ b(x) = (b_1(x), ..., b_k(x)).$$
(4)

Proof. The proof of continuity of h(x) in (3) goes along the same lines as the proof of Theorem 4 in 14.

Let us evaluate the operator $\mathcal{L}_u V$ for (1) using the feedback law u =h(x):

$$\mathcal{L}_h V = \begin{cases} a(x), & b = 0, \\ -(a^2(x) + \|b(x)\|^4)^{\frac{1}{2}}, \ b \neq 0, 2(a^2 + \|b\|^4)^{\frac{1}{2}} \ge \alpha(\|y\|), \\ -\frac{1}{2}\alpha(\|y\|), & \text{otherwise.} \end{cases}$$

As V(x) is a y-stochastic control Lyapunov function, the following inequality holds:

$$\mathcal{L}_h V \leq -\frac{1}{2} \alpha(||y||) \text{ for all } x \in \mathcal{D}_H.$$

Using Grönwall's inequality, we have:

$$E||y^{\xi,s}(t)||^2 \le k_1(t-s)e^{\int_s^t k_2 E||y^{\xi,s}(p)||^2 dp} \le N_1 e^{N_2 \delta^2},$$

where *E* is the expectation in the probability measure $P_{\xi,s}$, $y^{\xi,s}(t)$ is the *y*-component of the solution $x^{\xi,s}(t)$ of (2) with the initial data $x^{\xi,s}(s) = \xi$. Putting $\delta = ln(\frac{\epsilon_2\epsilon_1^2}{N_1})^{\frac{1}{2N_2}}$, we get

$$P\{sup_{t \ge t_0} ||y(t)|| > \epsilon_1\} \le \frac{E||y(t)||^2}{\epsilon_1^2} < \epsilon_2.$$

Let $\tau_{\varepsilon} = \inf\{t : \|y^{\xi,s}(t)\| > \varepsilon\}, \ \tau_{\varepsilon}(t) = \min(\tau_{\varepsilon},t).$ From Dynkin's lemma [15], it follows that

$$EV(x^{\xi,s}(\tau_{\varepsilon}(t))) - V(\xi) = E \int_{s}^{\tau_{\varepsilon}(t)} \mathcal{L}_{h}V(x^{\xi,s}(u))du.$$

Since $\mathcal{L}_h V(x) \leq -\frac{1}{2}\alpha(||y||)$, we will get

$$EV(x^{\xi,s}(\tau_{\varepsilon}(t))) \le V(\xi), \quad t \ge s.$$
(5)

The above inequality can be rewritten as

$$\int_{\tau_{\varepsilon} < t} \alpha_1(\|y^{\xi,s}(\tau_{\varepsilon})\|) P_{\xi,s}(d\omega) + \int_{\tau_{\varepsilon} \ge t} \alpha_1(\|y^{\xi,s}(t)\|) P_{\xi,s}(d\omega) \le V(\xi).$$

Hence, $\alpha_1(\varepsilon) P_{\xi,s} \{ \tau_{\varepsilon} < t \} \le V(\xi)$.

From the last equality, due to the continuity of the function V(x) and the equality V(0) = 0, it follows that

$$\lim_{\xi \to 0} P_{\xi,s} \{ \tau_{\varepsilon} < t \} = 0.$$

So, the equilibrium x = 0 of system (2) is y-stable in probability. From (5) it follows that the random process $V(x^{\xi,s}(\tau_{\varepsilon}(t)))$ is a nonnegative supermartingale, and there exists the limit

$$\lim_{t \to \infty} V(x^{\xi, s}(\tau_{\varepsilon}(t))) = \eta \tag{6}$$

with probability 1.

From the set of sample trajectories of the process $x^{\xi,s}(t)$ we take the subset B of sample trajectories such that for any $x_i^{\xi,s}(t)$ (i = 1, ..., n) the following equality holds: $\tau_{\varepsilon}(t) = t, t \in \mathbb{R}^+$. Then it follow from the above assumptions that

$$\lim_{\xi_y \to 0} P_{\xi,s}\{B\} = 1,$$
(7)

where $\xi^T = (\xi_y^T, \xi_z^T)$. From (6) and (7), we have

$$\lim_{t \to \infty} V(x^{\xi,s}(\tau_{\varepsilon}(t))) = \lim_{t \to \infty} V(x^{\xi,s}(t)) = \eta.$$
(8)

Note that V(x) is a y-stochastic control Lyapunov function, so for all trajectories from the set B, except a set of probability 0, the following property holds:

$$\lim_{t \to \infty} ||y^{\xi,s}(t)|| = 0.$$

214

From the assumption of z-extendability of solutions and (8), we obtain $\eta = 0$.

So, $\lim_{t\to\infty} ||y^{\xi,s}(t)|| = 0$. From this property it follows that the zero solution of the closed-loop system (2) is y-asymptotically stable in probability.

D Inverted pendulum with a moving mass

To illustrate possible applications of Theorem 3.1, we consider a mechanical system consisting of an inverted pendulum (carrier body) and a point mass m moving in the direction perpendicular to the axis of symmetry of the carrier body (Fig. 1). It is assumed that the mass m is suspended by a spring with the stiffness coefficient \varkappa .



Figure 1. Inverted pendulum with a moving mass.

We will use the following notations: M is the mass of the carrier body, φ is the angle between the axis of symmetry of the carrier and the vertical, y is the displacement of the point mass, and ℓ is the distance between the fixed point and the suspension of the mass m.

Let us first derive the equations of motion of this mechanical systems by using the Lagrangian formalism. The kinetic energy of the system is

$$T = \left(\frac{I}{2} + \frac{m(\ell^2 + y^2)}{2}\right)\dot{\varphi}^2 + \frac{m}{2}\dot{y}^2 + m\ell\dot{\varphi}\dot{y},$$

where I is the moment of inertia of the carrier body with respect to its

fixed point. The potential energy is

$$U = \frac{M\ell g}{2}\cos\varphi + \frac{\varkappa}{2}y^2 + mg(\ell\cos\varphi - y\sin\varphi).$$

Then the Lagrangian of the considered system takes the form

$$L = T - U = \left(\frac{I}{2} + \frac{m(\ell^2 + y^2)}{2}\right)\dot{\varphi}^2 + \frac{m}{2}\dot{y}^2 + m\ell\dot{\varphi}\dot{y} - \frac{\varkappa}{2}y^2 - \left(\frac{M}{2} + m\right)\ell g\cos\varphi + mgy\sin\varphi.$$

We now apply Lagrange's equations in the form

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{\varphi}} \right) - \frac{\partial L}{\partial \varphi} = 0, \\ \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{y}} \right) - \frac{\partial L}{\partial y} = F_u,$$

where F_u is the control force applied to the mass m. This leads to the following equations of motion:

$$\begin{split} \ddot{\varphi} &= \frac{1}{I + my^2} (-2my \dot{\varphi} \dot{y} - m\ell y \dot{\varphi}^2 + \varkappa \ell y + \frac{M\ell g}{2} \sin \varphi + \\ &+ mgy \cos \varphi - \ell F_u), \\ \ddot{y} &= \frac{\ell}{I + my^2} (2my \dot{\varphi} \dot{y} + m\ell y \dot{\varphi}^2 - \varkappa \ell y - \frac{M\ell g}{2} \sin \varphi + \\ &+ \ell F_u - mgy \cos \varphi) + \frac{I}{I + my^2} (y \dot{\varphi}^2 - \frac{\varkappa y}{m} + g \sin \varphi) + \\ &+ \frac{1}{I + my^2} ((\frac{I}{m} + y^2) F_u + my^3 \dot{\varphi}^2 + y^3 \varkappa + y^2 mg \sin \varphi). \end{split}$$

By replacing

$$v = \frac{1}{I + my^2} (-2my\dot{\varphi}\dot{y} - m\ell y\dot{\varphi}^2 + \varkappa\ell y + \frac{M\ell g}{2}\sin\varphi + mgy\cos\varphi - \ell F_u),$$

we obtain the following equations with respect to the new control v:

$$\begin{split} \ddot{\varphi} &= v, \\ \ddot{y} &= -(\ell + \frac{I + my^2}{m\ell})v + \frac{1}{I + my^2}(2y^3\varkappa + \frac{2m + M}{2m}(\frac{I}{m} + y^2)g\sin\varphi) - \frac{2y\dot{y}\dot{\varphi}}{\ell} + \frac{gy\cos\varphi}{\ell}. \end{split}$$

Let us rewrite the above equations of motion in the form $\dot{x} = f(x) + g(x)v$, where

$$x = \begin{bmatrix} x_1\\ x_2\\ x_3\\ x_4 \end{bmatrix} = \begin{bmatrix} \varphi\\ y\\ \dot{\varphi}\\ \dot{y} \end{bmatrix}, f(x) = \begin{bmatrix} x_3\\ x_4\\ 0\\ q(x) \end{bmatrix}, g(x) = \begin{bmatrix} 0\\ 0\\ 1\\ -\ell - \frac{I+my^2}{m\ell} \end{bmatrix}, \quad (9)$$

$$q(x) = \frac{1}{I + mx_2^2} \left(2x_2^3 \varkappa + \frac{2m + M}{2m} \left(\frac{I}{m} + x_2^2 \right) g \sin x_1 \right) - \frac{2x_2 x_3 x_4}{\ell} + \frac{g x_2 \cos x_1}{\ell}.$$

It is easy to see that system (9) admits the equilibrium x = 0 with v = 0 (upper equilibrium). We will consider the stabilization of the upper equilibrium of the carrier body in the sense of partial stabilization problem with respect to the variables (x_1, x_3) by applying control to the point mass.

To take into account random effects, we substitute the stochastic input $v = u + \lambda x_3 \dot{w}(t)$ formally into system (9), where w(t) is a standard onedimensional Wiener process. As a result, we obtain the following system of stochastic differential equations:

$$dx_{1} = x_{3}dt, dx_{2} = x_{4}dt, dx_{3} = udt + \lambda x_{3}dw(t), dx_{4} = \left(\left(-\ell - \frac{I + my^{2}}{m\ell} \right)u + q(x) \right) dt - \left(\ell + \frac{I + my^{2}}{m\ell} \right) \lambda x_{3}dw(t),$$
(10)

where u is treated as the control.

Since our goal is to steer the variables φ and $\dot{\varphi}$ (i.e. x_1 and x_3) to zero, we propose the following quadratic Lyapunov function candidate:

$$2V(x) = (k_1^2 + k_2^2 + k_2)x_3^2 + 2k_1x_1x_3 + (k_2 + 1)^2x_1^2,$$

where k_1 and k_2 are positive constants.

Let us define the functions a(x) and b(x) according to (4):

$$\begin{aligned} a(x) &= \sum_{i=1}^{4} f_i(x) \frac{\partial V(x)}{\partial x_i} + \frac{1}{2} \sum_{i,j=1}^{4} c_{ij}(x) \frac{\partial^2 V(x)}{\partial x_i \partial x_j} = \\ &= (k_1 x_3 + (k_2 + 1) x_1) x_3 + (k_2^2 + k_1^2 + k_2) \lambda^2 x_3^2, \\ &\quad b(x) = (k_2^2 + k_1^2 + k_2) x_3 + k_1 x_1. \end{aligned}$$

According to Theorem 3.1, we propose the feedback control law for system (10) in the form (3) with $\alpha(||y||) = \gamma ||y||^2$, $||y||^2 = x_1^2 + x_3^2$, $\gamma > 0$. So, the equilibrium x = 0 of the corresponding closed-loop system (10), (3) is asymptotically stable in probability with respect to (x_1, x_3) by Theorem 3.1. Simulation results for the closed-loop system (10), (3) with $k_1 = 2$, $k_2 = 1$ are presented in Fig. 2. These simulations have been performed in Maple by using the *ItoProcess*(·) function.



Figure 2. Components x_1 and x_3 of a sample path of the closed-loop system (10), (3).

E Stabilization of a three-wheeled trolley by a stochastic feedback law



Figure 3. Three-wheeled trolley.

Consider a mathematical model of the three-wheeled trolley whose position is determined by three coordinates: (x_1, x_2) are coordinates of the midpoint between the steering wheels, and x_3 is the angle between the axis of symmetry of the trolley and the x_1 -axis, cf. [19]. A cylindrical hinge whose axis is perpendicular to the axis of symmetry of the trolley is mounted above the point (x_1, x_2) (Fig. 3). A weightless and inextensible rod can rotate in this hinge, and a point mass is attached to the other end of the rod. We denote the angle between the vertical axis and the rod by α . The motion of the trolley is described by the rolling without slipping conditions:

$$dx_1 = (u_1 + u_2) \cos x_3 dt, dx_2 = (u_1 + u_2) \sin x_3 dt, dx_3 = (u_1 - u_2) dt,$$
(11)

where the vector $u = (u_1, u_2)^T \in \mathbb{R}^2$ is treated as the control.

Following 19, we also write Lagrange's equation with respect to the angle α :

$$\ddot{\alpha} - (\dot{x_1}\cos x_3 + \dot{x_2}\sin x_3 + \dot{x_3}\sin\alpha)\dot{x_3}\cos\alpha = -\sin\alpha.$$
(12)

Note that the considered model belongs to the class of nonholonomic systems which, as it is well-known, cannot be stabilized in a neighborhood of the equilibrium position by a deterministic continuous state feedback law (see, e.g., 20). In the sequel, we will study the stabilization problem with respect to a part of variables in the stochastic sense.

Let us denote the relative angular velocity of the rod by $\omega = \dot{\alpha}$ and perform the following change of variables in (11), (12):

 $\begin{aligned} z_1 &:= x_3, \\ z_2 &:= x_1 \cos x_3 + x_2 \sin x_3, \\ z_3 &:= x_1 \sin x_3 - x_2 \cos x_3, \\ z_4 &:= \alpha, \\ z_5 &:= \omega, \\ \nu_1 &:= u_1 - u_2, \\ \nu_2 &:= (u_1 + u_2) - (u_1 - u_2) z_3. \end{aligned}$

Then the equations of motion take the form:

$$\begin{aligned} \dot{z}_1 &= \nu_1, \\ \dot{z}_2 &= \nu_2, \\ \dot{z}_3 &= \nu_1 z_2, \\ \dot{z}_4 &= z_5, \\ \dot{z}_5 &= (\nu_2 + \nu_1 z_3 + \nu_1 \sin z_4) \nu_1 \cos z_4 - \sin z_4. \end{aligned}$$
(13)

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We randomize system (13) by designing the control inputs

$$\nu_1 = v_1, \nu_2 = v_2 + \lambda z_2 \dot{w}(t)$$

where $\dot{w}(t)$ is treated formally as the derivative of a standard one-dimensional Wiener process w(t). Then we rewrite the stochastic control system as follows:

$$dz_{1} = v_{1}dt, dz_{2} = v_{2}dt + \lambda z_{2}dw(t), dz_{3} = v_{1}z_{2}dt, dz_{4} = z_{5}dt, dz_{5} = ((v_{2} + v_{1}z_{3} + v_{1}\sin z_{4})v_{1}\cos z_{4} - \sin z_{4}) dt + +\lambda z_{2}v_{1}\cos z_{4}dw(t).$$
(14)

We consider the partial stabilization problem for system (14) with respect to the variables z_1, z_2, z_3 .

To design stabilizing controls v_1, v_2 , we take a control Lyapunov function candidate of the following form [21]:

$$V(z) = 2z_3 - \frac{1}{2}(z_1^2 + z_2^2)(1 + z_3^2) + 2\left(\frac{|z_1^2 + z_2^2|}{2}\right)^{1 + \frac{z_3^2}{2}}$$

Then we define the functions $a(z), b_1(z), b_2(z)$ according to (4):

$$a(z) = \frac{1}{2} \sum_{i,j=1}^{5} c_{ij}(z) \frac{\partial^2 V(z)}{\partial z_i \partial z_j} = \frac{1}{2} \lambda^2 z_2^2 \frac{\partial^2 V(z)}{\partial z_2^2},$$

$$\begin{split} b_1(z) &= -z_1(z_3^2+1) + \frac{4\left(\frac{|z_1^2+z_2^2|}{2}\right)^{1+\frac{z_3^2}{2}}(1+\frac{z_3^2}{2})z_1}{|z_1^2+z_2^2|} + z_2\left(2 - (z_1^2+z_2^2)z_3 + z_2\left(\frac{|z_1^2+z_2^2|}{2}\right)^{1+\frac{z_3^2}{2}}z_3\ln\left(\frac{|z_1^2+z_2^2|}{2}\right)\right), \\ &+ z_2\left(2 - (z_1^2+z_2^2)z_3 + 2\left(\frac{|z_1^2+z_2^2|}{2}\right)^{1+\frac{z_3^2}{2}}z_3\ln\left(\frac{|z_1^2+z_2^2|}{2}\right)\right), \\ &b_2(z) = -z_2(z_3^2+1) + \frac{4\left(\frac{|z_1^2+z_2^2|}{2}\right)^{1+\frac{z_3^2}{2}}(1+\frac{z_3^2}{2})z_2}{|z_1^2+z_2^2|}, \\ &b(z) = (b_1(z), b_2(z)). \end{split}$$

Thus, the conditions of Theorem 3.1 are satisfied with the above choice of a(x), b(x), and $\alpha(||y||) = \gamma ||y||^2$, $||y||^2 = z_1^2 + z_2^2 + z_3^2$, $\gamma > 0$. Numerical simulation results for system (14) with the feedback law (3) are presented in Figs. 4-5.



Figure 4. Components z_1, z_2, z_3 of a sample path of the closed-loop system (14), (3).



Figure 5. Components z_4, z_5 of a sample path of the closed-loop system (14), (3).

F Conclusion

A constructive proof of Artstein's theorem has been extended to the problem of partial stabilization of the Ito stochastic differential equations. This construction allows effective computing of stabilizing feedback controls if a control Lyapunov function in the sense of Definitions 2.1-2.2 is known. The control design scheme of Theorem 3.1 is shown to be applicable to nonlinear systems with stochastic effects that describe the dynamics of an inverted pendulum with a moving masses and a threewheeled trolley with an additional degree of freedom. The simulation results, presented in Figs. 2 and 4-5, illustrate the required behavior of sampled paths of the corresponding closed-loop systems.

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